

Indian Institute of Technology, Kharagpur

Date of Examination: 24 Nov 2017

Time: 3Hrs. Full Marks 100

End Semester Exam. 2015

3th Yr (MTech+Dual Degree) Thermal

Sub No. ME60017

Sub Name: **Conduction and Radiation**

No. of Students: 42

of the dept. of: **Mech Engg**

Answer all questions. Marks are given on the left margin in box
Wherever necessary, make suitable assumptions and state them clearly

1. Consider a one-dimensional transient heat conduction in a slab of thickness L . The slab is initially at constant temperature, T_0 , at time $t = 0$. The temperature of the face, $x = 0$, of the slab is suddenly increased to a constant value, T_1 , at time $t = 0$, and maintained at temperature T_1 for times $t > 0$, where $T_1 > T_0$. The temperature of the face, $x = L$, is kept at the constant value, T_0 , for times $t > 0$. The thermal diffusivity, α , of the slab may be assumed constant.
 - 2 (a) Determine the steady state temperature distribution, $T_{ss}(x)$, in the slab.
 - 4 (b) Formulate the initial boundary value problem for determining the temperature field, $T(x, t)$, in terms of the variable $u(x, t) = T(x, t) - T_{ss}(x)$. Obtain a partial differential equation for the function $u(x, t)$ and write the appropriate boundary and initial conditions for determining the solution, $u(x, t)$, of this equation in the domain $0 < x < L$ for $0 < t < \infty$.
 - 2 (c) Show that

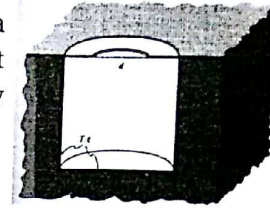
$$\frac{dI_1}{dt} = -I_2$$

where $I_1(t) = \int_0^L \frac{1}{2} [u(x, t)]^2 dx$ is the 'solution energy'

and $*I_2(t) = \alpha \int_0^L \frac{1}{2} \left[\frac{\partial u}{\partial x}(x, t) \right]^2 dx$ is the 'thermal dissipation'

- 2 (d) Use your answer to part(c) to show that $\lim_{t \rightarrow \infty} u(x, t) = 0$ for arbitrary initial conditions.
 - 12 (e) Use the method of *separation of variables* to determine the solution, $u(x, t)$, of the initial boundary value problem of part(b).
 - 2 (f) An Engineer thinks that steady state will be reached at time $t_f = K \frac{L^2}{\alpha}$, where K is a dimensionless constant. Do you agree with the engineer? Give reasons to justify your answer.
- 12 2. On an overcast day the directional behaviour of the intensity of solar radiation reaching the surface of the earth after being scattered by the atmosphere may be approximated as $I_{sky}(\theta) = I_{sky}(0) \times \cos\theta$, where θ is measured from the surface normal. For a day with $I_{sky}(0) = 100 \text{ W/m}^2 \cdot \text{sr}$ determine the solar radiation in W/m^2 hitting a solar collector that is horizontal. Neglect any radiation from the earth's surface hitting the solar collector.
 - 12 3. It is possible to manufacture a diffusely absorbing/emitting selective absorber with a spectral emissivity $\epsilon_\lambda = \epsilon_s = 0.05$ for $0 < \lambda < \lambda_c$ and $\epsilon_\lambda = \epsilon_c = 0.95$ for $\lambda > \lambda_c$, where λ_c is the cut-off wavelength. A solar collector has the above selective absorber plate with cut-off wavelength $\lambda_c = 3.0 \mu\text{m}$ that is at a temperature of 350 K and is exposed to solar radiation of $q_{sun} = 1000 \text{ W/m}^2$ at an angle of $\theta_s = 30^\circ$ off-normal. Determine the net radiative energy gain q_{net} in W/m^2 for such a solar collector. The temperature of the sun is 5800 K . The sky is assumed to be very cold.

4. A cylindrical cavity in a graphite block has a hole of diameter D and length L . The top of the cavity is covered with a disk, which has a hole of diameter d . The entire inside of the cavity is isothermal at temperature T , and grey, diffuse with emissivity ϵ . The entire cavity surface can be considered as a single zone.



- 8 (a) Determine the amount of radiation escaping from the cavity through the small hole.
 4 (b) What is the effective emissivity of the hole of diameter d if $d \ll D$ and $d \ll L$.
- 12 5. Two very large parallel isothermal plates of emissivity $\epsilon = 0.5$ and $\epsilon = 0.8$ exchange heat by radiation. Find the percentage reduction in radiation heat transfer when a polished aluminum plate of emissivity $\epsilon = 0.05$ is placed between them. Show the radiation network diagram. Assume diffuse gray surfaces.

6. Starting from the equation of radiative heat transfer show that

- 8 (a) The divergence of spectral radiation heat flux is

$$\nabla \cdot \mathbf{q}_\eta = \kappa_\eta (4\pi I_{b\eta} - G_\eta)$$

where, G is the *incident radiation function* and κ is the *absorption coefficient*

- 4 (b) The divergence of radiation heat flux for a grey medium is

$$\nabla \cdot \mathbf{q} = \kappa (4\sigma T^4 - G)$$

7. A 1mm thick slab of an absorbing/emitting gas has approximately linear temperature distribution between $T_1 = 1000\text{K}$ and $T_2 = 2000\text{K}$. On both sides, the medium is bounded by vacuum with non-reflecting boundaries.

- 4 (a) Write down the equation of radiative transport for the absorbing/emitting gray medium with no scattering.
 6 (b) If the medium has a constant and gray absorption coefficient of $\kappa = 1\text{m}^{-1}$, what is the radiative intensity (as a function of direction) leaving the hot side of the slab?
 6 (c) Give an expression for radiative heat flux leaving the hot side.

The fractional blackbody emissive power
The Stefan-Boltzmann Constant $\sigma = 5.67 \times 10^{-8} \text{W/m}^2 \cdot \text{K}^4$

$n\lambda T$ [μmK]	η/nT [cm^{-1}/K]	$E_{b\lambda}/n^3T^5$ [$\text{W/m}^2 \mu\text{mK}^5$]	$E_{b\eta}/nT^3$ [$\text{W/m}^2 \text{cm}^{-1}\text{K}^3$]	$f(n\lambda T)$
1000	10.0000	0.02110×10^{-11}	0.00211×10^{-8}	0.00032
1100	9.0909	0.04846	0.00586	0.00091
1200	8.3333	0.09329	0.01343	0.00213
1300	7.6923	0.15724	0.02657	0.00432
1400	7.1429	0.23932	0.04691	0.00779
1500	6.6667	0.33631	0.07567	0.01285
1600	6.2500	0.44359	0.11356	0.01972
1700	5.8824	0.55603	0.16069	0.02853
1800	5.5556	0.66872	0.21666	0.03934
1900	5.2632	0.77736	0.28063	0.05210
2000	5.0000	0.87858	0.35143	0.06672
2100	4.7619	0.96994	0.42774	0.08305
2200	4.5455	1.04990	0.50815	0.10088
2300	4.3478	1.11768	0.59125	0.12002
2400	4.1667	1.17314	0.67573	0.14025
2500	4.0000	1.21659	0.76037	0.16135
2600	3.8462	1.24868	0.84411	0.18311
2700	3.7037	1.27029	0.92604	0.20535
2800	3.5714	1.28242	1.00542	0.22788
2900	3.4483	1.28612	1.08162	0.25055
14,200	0.7042	0.03693	0.74456	0.96418
14,400	0.6944	0.03520	0.72998	0.96546
14,600	0.6849	0.03358	0.71579	0.96667
14,800	0.6757	0.03205	0.70198	0.96783
15,000	0.6667	0.03060	0.68853	0.96893
15,200	0.6579	0.02923	0.67544	0.96999
15,400	0.6494	0.02794	0.66270	0.97100
15,600	0.6410	0.02672	0.65029	0.97196
15,800	0.6329	0.02556	0.63820	0.97288
16,000	0.6250	0.02447	0.62643	0.97377
16,200	0.6173	0.02343	0.61496	0.97461
16,400	0.6098	0.02245	0.60379	0.97542
16,600	0.6024	0.02152	0.59290	0.97620
16,800	0.5952	0.02063	0.58228	0.97694
17,000	0.5882	0.01979	0.57194	0.97765
17,200	0.5814	0.01899	0.56186	0.97834
17,400	0.5747	0.01823	0.55202	0.97899
17,600	0.5682	0.01751	0.54243	0.97962
17,800	0.5618	0.01682	0.53308	0.98023
18,000	0.5556	0.01617	0.52396	0.98081
18,200	0.5495	0.01555	0.51506	0.98137
18,400	0.5435	0.01496	0.50638	0.98191
18,600	0.5376	0.01439	0.49790	0.98243
18,800	0.5319	0.01385	0.48963	0.98293