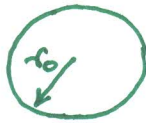


2.



shaft

When the shaft rotates alone,

$$\sigma_r = 0 \quad \text{at} \quad r = r_0, \quad r_0 > a$$

$$\therefore -\frac{3+\nu}{8} \rho \omega^2 r_0^2 + A = 0 \quad [\text{for solid shaft } B=0]$$

$$\Rightarrow A = \frac{3+\nu}{8} \rho \omega^2 r_0^2$$

$$\therefore \sigma_r|_{\text{shaft}} = \frac{3+\nu}{8} \rho \omega^2 (r_0^2 - r^2)$$

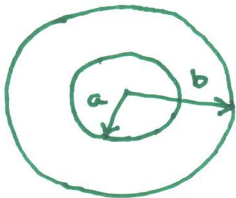
$$\& \sigma_\theta|_{\text{shaft}} = -\frac{1+3\nu}{8} \rho \omega^2 r^2 + \frac{3+\nu}{8} \rho \omega^2 r_0^2$$

$$\epsilon_\theta|_{\text{shaft}} = \frac{u_r}{r_0} = \frac{1}{E} (\sigma_\theta - \nu \sigma_r)$$

$$\begin{aligned} \therefore u_r(r_0)|_{\text{shaft}} &= \frac{r_0}{E} \left[-\frac{1+3\nu}{8} \rho \omega^2 r_0^2 + \frac{3+\nu}{8} \rho \omega^2 r_0^2 - \nu \cdot 0 \right] \\ &= \frac{\rho(1-\nu)}{4E} \omega^2 r_0^3 \end{aligned}$$

When the disc rotates alone,

$$\sigma_r = 0 \quad \text{at} \quad r = a \quad \text{and} \quad b$$



disc ($r_0 > a$)

$$\therefore -\frac{3+\nu}{8} \rho \omega^2 a^2 + A_1 + \frac{B_1}{a^2} = 0$$

$$-\frac{3+\nu}{8} \rho \omega^2 b^2 + A_1 + \frac{B_1}{b^2} = 0$$

$$\Rightarrow A_1 = \frac{3+\nu}{8} \rho \omega^2 (a^2 + b^2)$$

$$B_1 = -\frac{3+\nu}{8} \rho \omega^2 a^2 b^2$$

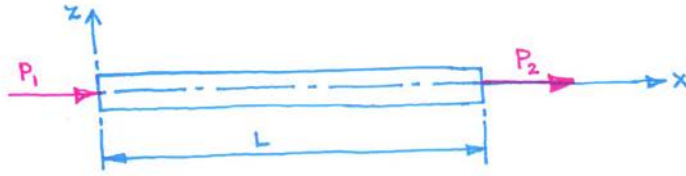
$$\begin{aligned} \therefore \sigma_\theta(a)|_{\text{disc}} &= -\frac{1+3\nu}{8} \rho \omega^2 a^2 + \frac{3+\nu}{8} \rho \omega^2 (a^2 + b^2) + \frac{3+\nu}{8} \rho \omega^2 b^2 \\ &= \frac{\rho \omega^2 (1-\nu)}{4} a^2 + \frac{3+\nu}{4} \rho \omega^2 b^2 \end{aligned}$$

$$\begin{aligned} u_r(a)|_{\text{disc}} &= a \epsilon_\theta(a)|_{\text{disc}} = \frac{a}{E} (\sigma_\theta - \nu \sigma_r)|_{r=a, \text{disc}} \\ &= \frac{\rho \omega^2 (1-\nu) a^3}{4E} + \frac{3+\nu}{4E} \rho \omega^2 a b^2 \end{aligned}$$

\therefore The radial interference, δ is given by

$$\begin{aligned} \delta &= r_0 + u_r(r_0)|_{\text{shaft}} - (a + u_r(a)|_{\text{disc}}) \\ &= \frac{\rho(1-\nu)\omega^2}{4E} (r_0^3 - a^3) - \frac{3+\nu}{4E} \rho \omega^2 a b^2 + r_0 - a \quad \text{Proved} \end{aligned}$$

Buckling of Column



For equilibrium: $P_2 = -P_1$

Let $|P_1| = P \quad \therefore P_2 = -P$

$$P_1 \rightarrow \text{column} \leftarrow P_2 \Rightarrow \int_A \sigma_x dA + P_1 = 0 \Rightarrow \int_A \sigma_x dA = -P_1 = -P$$

Displacement field: $u_0 - z \frac{\partial w_0}{\partial x}$, $w = w_0$

$$\epsilon_x = \frac{\partial u}{\partial x} + \frac{1}{2} \left(\frac{\partial w}{\partial x} \right)^2 = \frac{\partial u_0}{\partial x} + \frac{1}{2} \left(\frac{\partial w_0}{\partial x} \right)^2 - z \frac{\partial^2 w_0}{\partial x^2}, \quad \sigma_x = E \epsilon_x$$

Internal virtual work, $\delta U = \int_{\Omega} \sigma_x \delta \epsilon_x d\Omega$

$$\text{or, } \delta U = \int_0^L \int_A \sigma_x \left[\delta \left(\frac{\partial u_0}{\partial x} \right) + \frac{1}{2} \delta \left(\frac{\partial w_0}{\partial x} \right)^2 - z \delta \left(\frac{\partial^2 w_0}{\partial x^2} \right) \right] dA dx$$

$$= \int_0^L \int_A \sigma_x \left[\delta \left(\frac{\partial u_0}{\partial x} \right) + \frac{\partial w_0}{\partial x} \delta \left(\frac{\partial w_0}{\partial x} \right) \right] dA dx$$

$$- \int_0^L \int_A \sigma_x z \delta \left(\frac{\partial^2 w_0}{\partial x^2} \right) dA dx$$

$$= - \int_0^L P \left[\delta \left(\frac{\partial u_0}{\partial x} \right) + \frac{\partial w_0}{\partial x} \delta \left(\frac{\partial w_0}{\partial x} \right) \right] dx$$

$$- \int_0^L \int_A E \left[\frac{\partial u_0}{\partial x} + \frac{1}{2} \left(\frac{\partial w_0}{\partial x} \right)^2 - z \frac{\partial^2 w_0}{\partial x^2} \right] z \delta \left(\frac{\partial^2 w_0}{\partial x^2} \right) dA dx$$

$$= - \int_0^L P \left[\delta \left(\frac{\partial u_0}{\partial x} \right) + \frac{\partial w_0}{\partial x} \delta \left(\frac{\partial w_0}{\partial x} \right) \right] dx + EI \int_0^L \frac{\partial^2 w_0}{\partial x^2} \delta \left(\frac{\partial^2 w_0}{\partial x^2} \right) dx$$

\therefore x -axis is a centroidal axis, $\int_A z dA = 0$, $\int_A z^2 dA = I$

$$\text{Now, } - \int_0^L P \delta \left(\frac{\partial u_0}{\partial x} \right) dx = - \int_0^L P \frac{\partial}{\partial x} (\delta u_0) dx = -P \delta u_0 \Big|_0^L + \int_0^L \frac{\partial P}{\partial x} \delta u_0 dx$$

$[\because P \text{ is const.}]$

$$-\int_0^L P \frac{\partial W_0}{\partial x} \delta \left(\frac{\partial W_0}{\partial x} \right) dx = -\int_0^L P \frac{\partial W_0}{\partial x} \frac{\partial}{\partial x} (\delta W_0) dx$$

$$= -P \frac{\partial W_0}{\partial x} \delta W_0 \Big|_0^L + \int_0^L P \frac{\partial^2 W_0}{\partial x^2} \delta W_0 dx$$

$$EI \int_0^L \frac{\partial^2 W_0}{\partial x^2} \delta \left(\frac{\partial^2 W_0}{\partial x^2} \right) dx = EI \int_0^L \frac{\partial^2 W_0}{\partial x^2} \frac{\partial}{\partial x} \left\{ \delta \left(\frac{\partial W_0}{\partial x} \right) \right\} dx$$

$$= EI \frac{\partial^2 W_0}{\partial x^2} \delta \left(\frac{\partial W_0}{\partial x} \right) \Big|_0^L - EI \int_0^L \frac{\partial^3 W_0}{\partial x^3} \delta \left(\frac{\partial W_0}{\partial x} \right) dx$$

$$= EI \frac{\partial^2 W_0}{\partial x^2} \delta \left(\frac{\partial W_0}{\partial x} \right) \Big|_0^L - EI \frac{\partial^3 W_0}{\partial x^3} \delta W_0 \Big|_0^L + EI \int_0^L \frac{\partial^4 W_0}{\partial x^4} \delta W_0 dx$$

External virtual work: $\delta V = -P_1 \delta u_0(0) - P_2 \delta u_0(L)$
 $= -P \delta u_0(0) + P \delta u_0(L)$

Virtual work principle: $\delta \Pi = \delta U + \delta V = 0$ yields

$$-P \delta u_0(L) + P \delta u_0(0) - P \frac{\partial W_0}{\partial x} \delta W_0 \Big|_0^L + EI \frac{\partial^2 W_0}{\partial x^2} \delta \left(\frac{\partial W_0}{\partial x} \right) \Big|_0^L$$

$$- EI \frac{\partial^3 W_0}{\partial x^3} \delta W_0 \Big|_0^L + \int_0^L \left(EI \frac{\partial^4 W_0}{\partial x^4} + P \frac{\partial^2 W_0}{\partial x^2} \right) \delta W_0 dx - P \delta u_0(0) + P \delta u_0(L) = 0$$

\therefore The governing equation is

$$EI \frac{\partial^4 W_0}{\partial x^4} + P \frac{\partial^2 W_0}{\partial x^2} = 0 \quad \text{or,} \quad \frac{\partial^4 W_0}{\partial x^4} + \lambda^2 \frac{\partial^2 W_0}{\partial x^2} = 0, \quad \lambda^2 = \frac{P}{EI}$$

Boundary conditions for fixed-free column:

$$W_0 = \frac{\partial W_0}{\partial x} = 0 \quad \text{at } x = 0$$

$$\frac{\partial^2 W_0}{\partial x^2} = 0 \quad \text{and} \quad \frac{\partial^3 W_0}{\partial x^3} + \lambda^2 \frac{\partial W_0}{\partial x} = 0 \quad \text{at } x = L$$

general solution of the governing equation:

$$W_0 = c_1 \sin \lambda x + c_2 \cos \lambda x + c_3 x + c_4$$

The boundary conditions yield the following equations:

$$c_2 + c_4 = 0 \quad \text{--- (1)} \quad c_1 \lambda + c_3 = 0 \quad \text{--- (2)}$$

$$c_1 \sin \lambda L + c_2 \cos \lambda L = 0 \quad \text{--- (3)} \quad \text{and} \quad c_3 = 0 \quad \text{--- (4)}$$

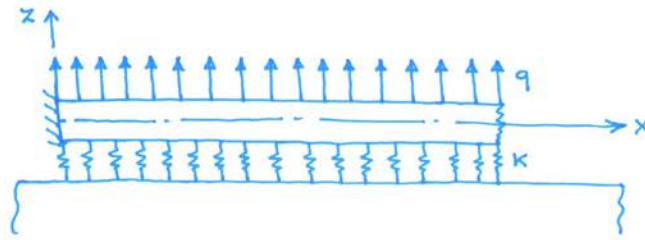
\therefore from (2), $c_3 = 0$ from (3) $c_2 \cos \lambda L = 0$. If $c_2 = 0$, $c_1 = 0$ \therefore Buckling will not occur. For buckling $c_2 \neq 0$ and $\cos \lambda L = 0$ this yields

$$\lambda L = \frac{n\pi}{2}, \quad n = 1, 3, 5, \dots \quad \text{or,} \quad \lambda^2 = \frac{n^2 \pi^2}{4L^2} \quad \text{or} \quad \frac{P}{EI} = \frac{n^2 \pi^2}{4L^2}$$

P will be minimum if $n=1$. \therefore the critical buckling load

$$\text{is } P_{cr} = \frac{\pi^2 EI}{4L^2}$$

Semi-infinite beam on elastic foundation



$$u = -z \frac{\partial w_0}{\partial x}, \quad w = w_0, \quad \epsilon_x = -z \frac{\partial^2 w_0}{\partial x^2}$$

$$\begin{aligned} \delta U_1 &= \int_0^{\infty} \int_A \sigma_x \delta \epsilon_x dA dx = \int_0^{\infty} \int_A E \frac{\partial^2 w_0}{\partial x^2} z^2 \delta \left(\frac{\partial^2 w_0}{\partial x^2} \right) dA dx \\ &= \int_0^{\infty} EI \frac{\partial^2 w_0}{\partial x^2} \delta \left(\frac{\partial^2 w_0}{\partial x^2} \right) dx = EI \frac{\partial^2 w_0}{\partial x^2} \delta \left(\frac{\partial w_0}{\partial x} \right) \Big|_0^{\infty} - \int_0^{\infty} EI \frac{\partial^3 w_0}{\partial x^3} \delta \left(\frac{\partial w_0}{\partial x} \right) dx \\ &= EI \frac{\partial^2 w_0}{\partial x^2} \delta \left(\frac{\partial w_0}{\partial x} \right) \Big|_0^{\infty} - EI \frac{\partial^3 w_0}{\partial x^3} \delta w_0 \Big|_0^{\infty} + \int_0^{\infty} EI \frac{\partial^4 w_0}{\partial x^4} \delta w_0 dx \end{aligned}$$

$$\delta U_2 = \frac{1}{2} \int_0^{\infty} k \delta (w_0)^2 dx = \int_0^{\infty} k w_0 \delta w_0 dx$$

$$\delta V = - \int_0^{\infty} q \delta w_0 dx$$

virtual work principle: $\delta \Pi = \delta U_1 + \delta U_2 + \delta V = 0 \Rightarrow$

$$EI \frac{\partial^2 w_0}{\partial x^2} \delta \left(\frac{\partial w_0}{\partial x} \right) \Big|_0^{\infty} - EI \frac{\partial^3 w_0}{\partial x^3} \delta w_0 \Big|_0^{\infty} + \int_0^{\infty} \left(EI \frac{\partial^4 w_0}{\partial x^4} + k w_0 - q \right) \delta w_0 dx = 0$$

\Rightarrow Governing equation:

$$EI \frac{\partial^4 w_0}{\partial x^4} + k w_0 = q$$

Boundary conditions: $w_0 = \frac{\partial w_0}{\partial x} = 0$ at $x = 0$

$$\frac{\partial^2 w_0}{\partial x^2} = \frac{\partial^3 w_0}{\partial x^3} = 0 \text{ at } x = \infty$$

General Solution: $w_0 = B_1 e^{\beta x} \cos \beta x + B_2 e^{\beta x} \sin \beta x + B_3 e^{-\beta x} \cos \beta x + B_4 e^{-\beta x} \sin \beta x$

$B_1 = B_2 = 0$ because of $\frac{\partial^2 w_0}{\partial x^2} = \frac{\partial^3 w_0}{\partial x^3} = 0$ at $x = \infty$

$$w_0 = 0 \text{ at } x = 0 \Rightarrow B_3 + \frac{q}{k} = 0 \therefore B_3 = -\frac{q}{k}$$

$$\frac{\partial w_0}{\partial x} = 0 \text{ at } x = 0 \Rightarrow -B_3 \beta + B_4 \beta = 0 \therefore B_4 = B_3 = -\frac{q}{k}$$

∴ The solution of deflection is

$$w_0 = -\frac{q}{k} [e^{-\beta x} (\cos \beta x + \sin \beta x) - 1]$$

For maximum bending moment it must be that

$$\frac{dM}{dx} = 0 \text{ and } \frac{d^2M}{dx^2} < 0 \text{ at a location.}$$

Now,  $\Rightarrow -M_1 + M = 0$ or, $M_1 = M$

Also  $\Rightarrow -M_1 - \int_A q_x dA \cdot z = 0$

$$\text{or, } M_1 = - \int_A E E_x z dA = \int_A E z^2 \frac{\partial^2 w_0}{\partial x^2} dA = E \frac{\partial^2 w_0}{\partial x^2} \int_A z^2 dA$$

$$\therefore M = EI \frac{\partial^2 w_0}{\partial x^2}$$

Thus $\frac{dM}{dx} = 0 \Rightarrow \frac{\partial^3 w_0}{\partial x^3} = 0 \Rightarrow -\frac{q\beta^3}{k} e^{-\beta x} \cos \beta x = 0$

∴ $x = \infty$ is not feasible ∴ $\cos \beta x = 0$

$$\text{or } \beta x = \frac{n\pi}{2}, \quad n = 1, 3, 5, \dots$$

Now, $\frac{d^2M}{dx^2} = EI \frac{\partial^4 w_0}{\partial x^4} = -k w_0 + q = +q [e^{-\beta x} (\cos \beta x + \sin \beta x) - 1] + q$
 $= q e^{-\beta x} (\cos \beta x + \sin \beta x)$

Obviously, $\frac{d^2M}{dx^2}$ will be < 0 first if $\beta x = \frac{3\pi}{2}$

For the next higher values of βx such that $\frac{d^2M}{dx^2} < 0$, βx will be > 4 which yields decayed result.

∴ M is maximum at $\beta x = \frac{3\pi}{2}$

$$\begin{aligned} \therefore M_{\max} &= EI \cdot \frac{2\beta^2 q}{k} e^{-3\pi/2} \left[\cos\left(\frac{3\pi}{2}\right) - \sin\left(\frac{3\pi}{2}\right) \right] \\ &= \frac{2EIq\beta^2}{k} e^{-3\pi/2} \end{aligned}$$

* Care must be taken if q is downward.